

# \* IMPULSE MOMENTUM EQUATIONS

Moment of momentum :- (Torque)

$$P = \frac{2\pi NT}{60}$$

$m$  = mass flow rate

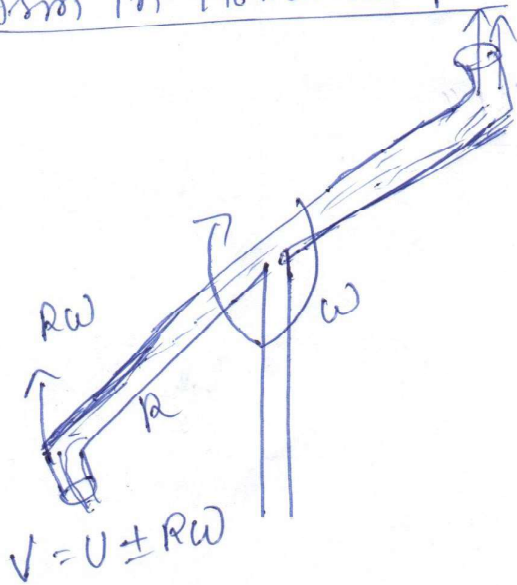
$R$  = radius

$v$  = velocity

$$T = \frac{d}{dt} [m v r]$$

Lawn sprinkles Application

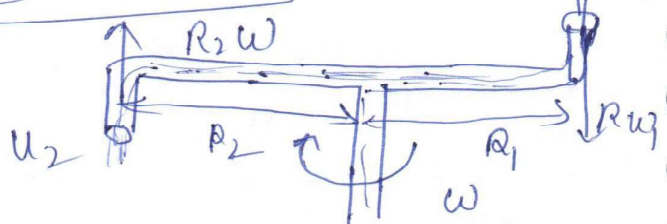
Asm in horizontal plane



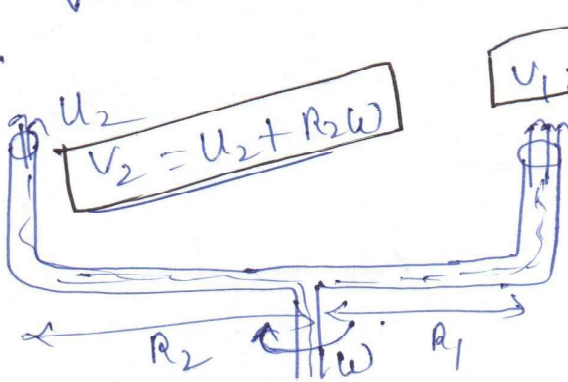
$U$  = velocity of flow in ~~the~~ (radius)

$$V_2 = U_2 + R\omega$$

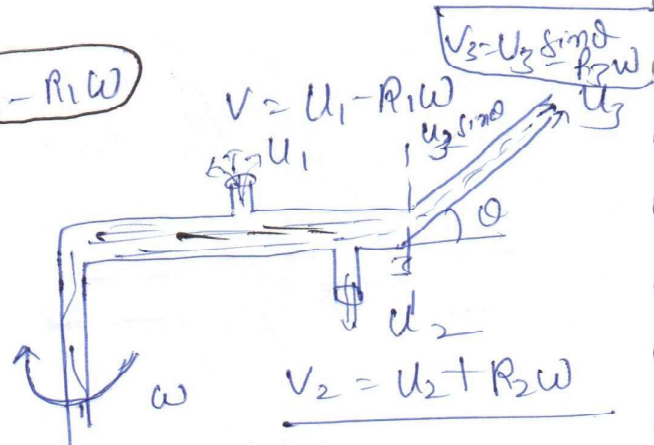
$$V_1 = U_1 - R\omega$$



$$T = \dot{m}_1 V_1 R + \dot{m}_2 V_2 R_2$$



$$T = +\dot{m}_1 V_1 R_1 - \dot{m}_2 V_2 R_2$$



$$T = +\dot{m}_1 V_1 R_1 - \dot{m}_2 V_2 R_2 + \dot{m}_3 V_3 R$$

# Impact of jet

① on flat, smooth, vertical fixed

plate / blade

cone / resultant

1) Flat plate

power = 0

P = pressure

$F_n$  = Force on the plate

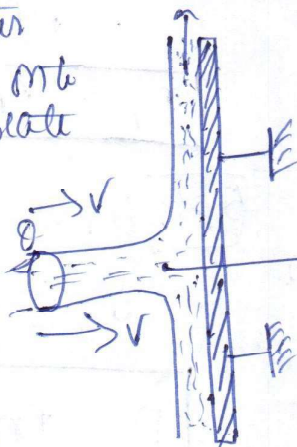
∝ Impact of water

∝ Energy is forced into the plate

∝  $F = ma$

$= m \times \frac{A \cdot vel}{time}$

$= \frac{m}{time} \times (A \cdot vel)$



$F_n$  (Normal Force)  
Force on the vertical plate

Power =  $F_n \times vel$

$P = F_n \times u^0$

$P = 0$

$F_n = \dot{m}(A \cdot vel)_n = \rho A (v_1 - v_2)_n$

$P = F_n \times v_e$  (blade)  
 $u =$  blade velocity

$P = F_n \times u$

$\rightarrow F_n = \dot{m} (A \cdot vel)$

$= \rho A v (v - 0)$

$F_n = \rho A v^2$

$\rightarrow \frac{P}{Power} = F_n \times u^0 = 0$

∝ The flow is transformed into tangential

|                   |              |
|-------------------|--------------|
| $F_n$             | $\rho A v^2$ |
| $\frac{Power}{0}$ |              |

Force but i.e. power '0'

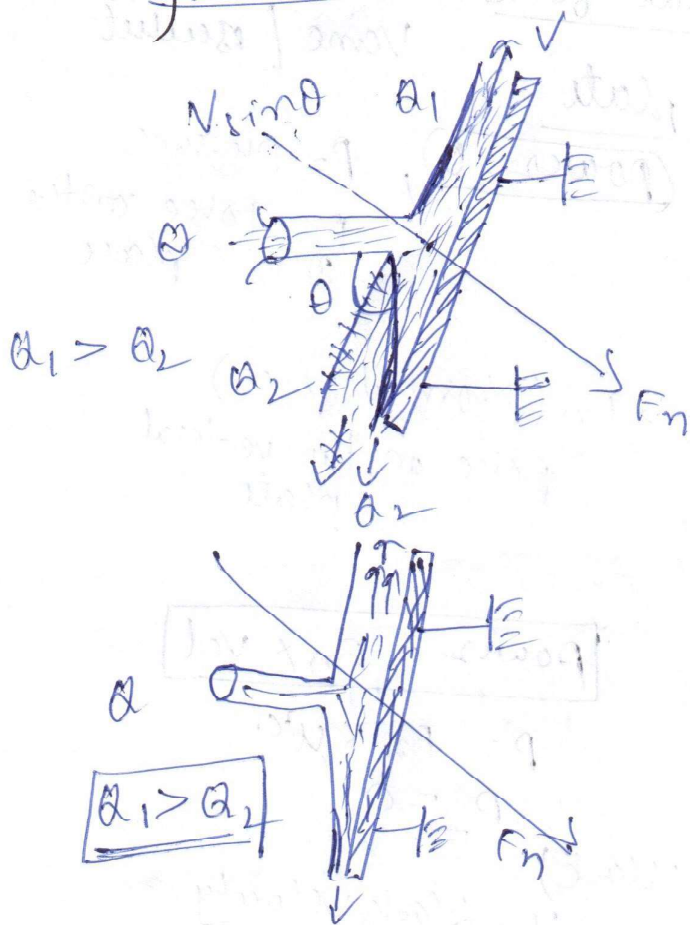
$F = ma$

$w = F \times dist$

$P = \frac{work}{time} = \frac{F \times distance}{time}$

$P = F \times vel$

2) Inclined plane :-



$$F_n = \dot{m} (A v \sin \theta)_n$$

$$= \rho A V (v \sin \theta - 0)$$

After impact

$$F_n = \rho A V^2 \sin \theta$$

$$\alpha_1 = \alpha_2 [1 + \cos \theta]$$

$$\alpha_2 = \alpha_1 [1 - \cos \theta]$$

$$P = F_n \times u$$

$u$  = velocity of the plate in the direction of jet

3) curved surface plane :- (Symmetric)

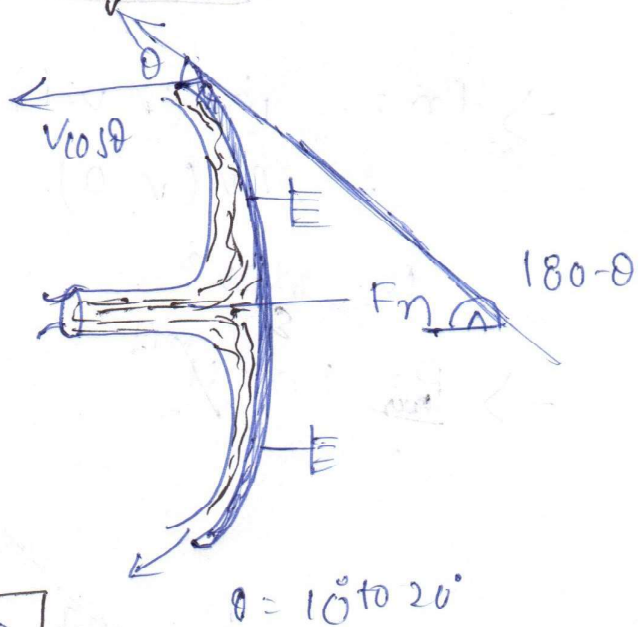
Deflection angle optimum  $160^\circ$  to  $170^\circ$

$$F_n = \dot{m} (A v \sin \theta)_n$$

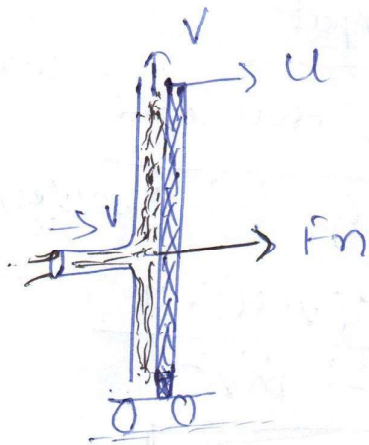
$$F_n = \rho A V (v - (-v \cos \theta))$$

Power  $P = F_n \times u = 0$

$$F_n = \rho A V^2 [1 + \cos \theta]$$



4) Moving plate :-  $u < v$



$$F_n = \dot{m} (\Delta v)_{\dot{m}}$$

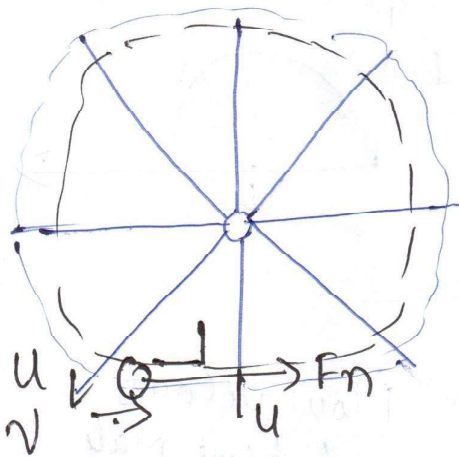
$$F_n = \rho A (v-u) (v-u)$$

$$F_n = \rho A (v-u)^2$$

$$P = F_n \times u$$

$$P = \rho A (v-u)^2 \cdot u$$

5) Series of blades :-



→ In series of blades  
are connected then  
→  $(v-u) + x_i$   
→ Blade is ~~not~~ connect  
with only one at a time  
all other not hit  
but rotated

$$F_n = \dot{m} (\Delta v)_{\dot{m}}$$

$$= \rho A v [(v-u) - 0]$$

$$F_n = \rho A v [v-u]$$

$$P = F_n \times u = \rho A v [v-u] \cdot u$$

$$\eta_{\text{transmission}} = \frac{\text{output}}{\text{input}}$$

$$\text{K.E of jet} = \frac{1}{2} \frac{mv^2}{\text{time}}$$

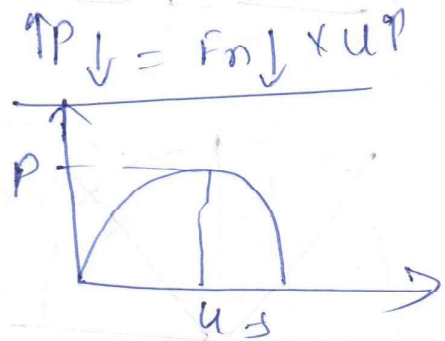
$$= \frac{\text{power developed}}{\text{K.E of jet (Joules/sec)}} \rightarrow \text{watt}$$

$$\frac{\frac{1}{2} mv^2}{\dots} = \frac{\rho A v (v-u) \cdot u}{\frac{1}{2} mv^2}$$

$$\eta = \frac{2u(v-u)}{v^2}$$

Condition for  $\eta_{\text{max}}$   
 $u = v/2$

$$\eta = \frac{2u(v-u)}{v^2}$$



$$\frac{d\eta}{du} = 0 \frac{d}{du} \left[ \frac{2(vu - u^2)}{v^2} \right] = 0$$

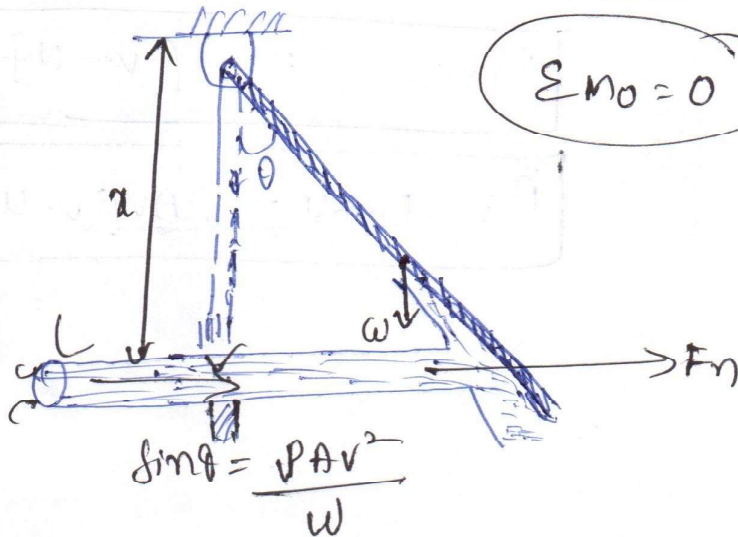
$$\frac{2}{v^2} [v - 2u] = 0$$

$$u = v/2$$

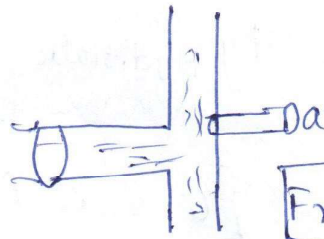
Flow velocity of vent plate

$$\text{But} = \frac{\rho A v (v-u) u \pi}{\text{as } u \uparrow \Rightarrow \text{in } \downarrow}$$

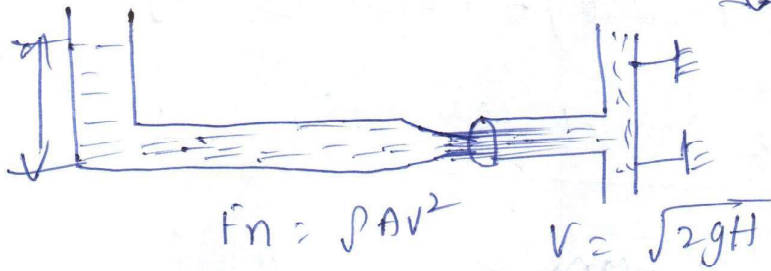
Sp. case:-



$$\sin \theta = \frac{2 \rho A v^2 \cdot x}{w \cdot L}$$



$$F_n = \rho(A-a)v^2$$

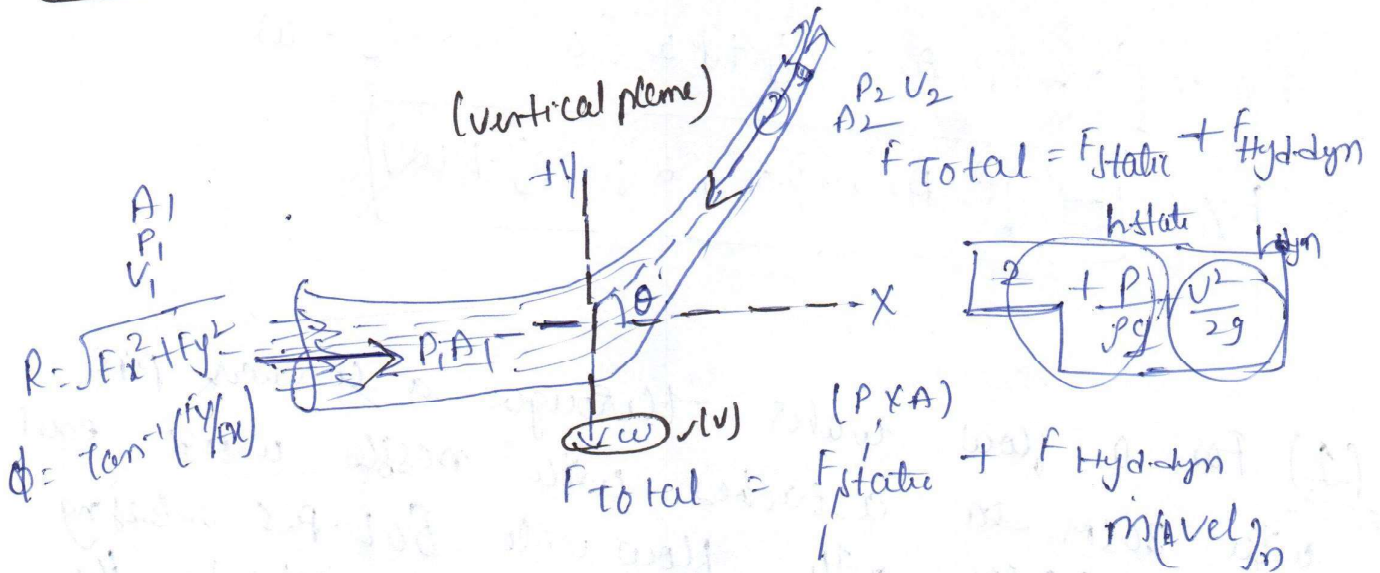


$$F_n = \rho A v^2$$

$$v = \sqrt{2gH}$$

→ Force in pipe bend ←

Horizontal plane (Reducing bend)



$$R = \sqrt{F_x^2 + F_y^2}$$

$$\phi = \tan^{-1}(F_y/F_x)$$

$$F_{Total} = F_{Static} + F_{Hyd-dyn}$$

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m(Avel)<sub>2</sub>

$$F_x Total = (F_{xHyd-dyn}) + (F_{xHyd-dyn})_m(Avel)_x$$

$$F_x = [P_1 A_1 - P_2 A_2 \cos \theta]$$

$$F_x = [P_1 A_1 - P_2 A_2 \cos \theta] \Rightarrow F_x = [P_1 A_1 - P_2 A_2 \cos \theta]$$

$$\rightarrow F_x = [P_1 A_1 - P_2 A_2 \cos \theta] + \rho Q [v_1 - (v_2 \cos \theta)]$$

$$F_{y \text{ total}} = F_{y \text{ Hydrostatic}} + F_{y \text{ Hydrodynamic}} \quad P_1 A_1 \quad P_2 A_2$$

$$F_y = [0 - P_2 A_2 \sin \theta] + \rho a [0 - v_2 \sin \theta]$$

$$F_y = - [P_2 A_2 \sin \theta + \rho a v_2 \sin \theta]$$

→ vertical plane ←

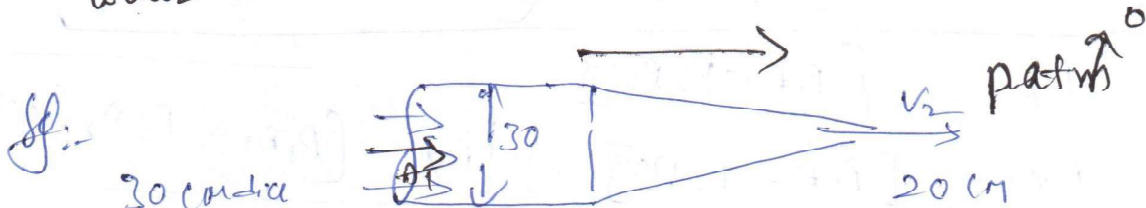
$$F_x = [P_1 A_1 - P_2 A_2 \cos \theta] + \rho a [v_1 - (v_2 \cos \theta)]$$

do same

$$F_y = [0 - P_2 A_2 \sin \theta] + \rho a [0 - v_2 \sin \theta] - W$$

$$F_y = - [P_2 A_2 \sin \theta + \rho a v_2 \sin \theta + W]$$

(1) For a flow water through a circular pipe with 30 cm dia attached into nozzle whose exit dia is 20 cm with flow rate 50 L.P.S releasing into atmosphere. Find the force exerted by the water on the nozzle



$$F_{\text{Total}} = F_{\text{Hyd. Static}} + F_{\text{Hyd. dyn}}$$

$$= [P_1 A_1 - P_2 A_2] + \rho a [v_1 - v_2]$$

$$\left[ z_1 + \frac{p}{\rho g} + \frac{v_1^2}{2g} \right]_1 = \left[ z_1 + \frac{p}{\rho g} + \frac{v_1^2}{2g} \right]$$

$$\frac{p_1}{\rho g} = \frac{v_2^2 - v_1^2}{2g} = \frac{\left( \frac{Q}{A_2} \right)^2 - \left( \frac{Q}{A_1} \right)^2}{2}$$

$$p_1 = \frac{Q^2}{2} \cdot \rho \left[ \frac{1}{A_2^2} - \frac{1}{A_1^2} \right]$$

$$p_1 = \frac{(50 \times 10^3)^2}{2} \cdot 10^3 \left[ \frac{1}{(0.0314)^2} - \frac{1}{(0.0707)^2} \right]$$

$$p_1 = 1.016 \text{ kPa}$$

Then  $(p_1 A_1) + \rho Q^2 [v_1 - v_2]$

$$= (1016 \times 0.0707) + 10^3 (50 \times 10^3)^2 \left[ \frac{1}{0.0707} - \frac{1}{0.0314} \right]$$

$$F_T = 27.6 \text{ N} \quad \checkmark$$

Self learning

- pumps
- performance curves
- Hydraulic turbines